## بسـم الله الرحمن الرحيم

King Abdulaziz University<br>Engineering College<br>Department of Production and Mechanical System Design



MENG 470 Mechanical Vibrations
First Exam Solution
Closed-book Exam
Monday: 8/2/1425 H
Time Allowed: 90 mins

| Question 1 | 5 | 5 |
| :---: | :---: | :---: |
| Question 2 | 5 | 5 |
| Question 3 | 5 | 5 |
| TOTAL | 15 | 15 |

1. There are totally 3 problems in this exam.
2. This is a closed book and closed notes Opportunity to Shine
3. Show all work for partial credit.
4. Assemble your work for each problem in logical order.
5. Justify your conclusion. I cannot read minds.

## بسـم الله الرحمن الرحيم

Mechanical Vibrations
MENG 470
First Exam

Closed Book Exam
Time: 1 Hour
Monday: 8/2/1425 H

1. The single degree-of-freedom harmonic oscillator shown below in Figure 1 has a logarithmic decrement and static deflection equal to 1.435 and 0.765 m , respectively. The spring stiffness is $k=525 \mathrm{~N} / \mathrm{m}$.
(a) Show the characteristic roots of the system in the complex plane.
(b) Indicate the free response type (i.e., undamped, underdamped, critically damped, or overdamped).
(c) If the oscillator is subjected to a (vertical) harmonic force with magnitude $f_{o}$ and frequency 0.477 Hz , determine the range of $f_{o}$ such that the steady-state amplitude of vibration is less than 0.798 cm .


Figure 1
Sol.:
(a)

$$
\omega_{n}=\sqrt{\frac{g}{\delta_{s t}}}=\sqrt{\frac{9.81}{0.765}}=3.581 \mathrm{rad} / \mathrm{sec}
$$

where

$$
\begin{aligned}
\zeta & =\frac{\delta}{\sqrt{(2 \pi)^{2}+\delta^{2}}}=\frac{1.435}{\sqrt{(2 \pi)^{2}+1.435^{2}}}=0.223 \\
\omega & =2 \pi f=2 \pi(0.477)=2.997 \mathrm{rad} / \mathrm{sec}
\end{aligned}
$$

The characteristics roots are:

$$
D_{1,2}=-\zeta \omega_{n} \pm \omega_{n} \sqrt{\zeta^{2}-1}=-(0.223)(3.581) \pm(3.581) \sqrt{(0.223)^{2}-1}=-0.797 \pm i 3.49
$$


(b) Since the dumping ratio $=0.22$ (i.e. $0<\zeta<1$ ), then the system is underdamped.
(c) $f=0.477 \mathrm{~Hz}, X_{\max }=0.00798 \mathrm{~m}$,
$\omega_{n}=3.581 \mathrm{rad} / \mathrm{sec}, \quad k=525 \mathrm{Nm}, \quad$ and $\quad \zeta=0.22$
$m=\frac{k}{\omega_{n}{ }^{2}}=\frac{525}{3.581^{2}}=40.94 \mathrm{~kg}$
$c=2 \zeta \omega_{n} m=2(0.223)(3.581)(40.94)=65.39 \mathrm{rad} . \mathrm{kg} / \mathrm{sec}$
$\omega=0.477(2 \pi)=2.997 \mathrm{rad} / \mathrm{sec}$
$\left|\left(f_{0}\right)_{\max }\right|=X \sqrt{\left(k-m \omega^{2}\right)+(c \omega)^{2}}$
$\left|\left(f_{0}\right)_{\max }\right|=0.00798 \sqrt{\left(525-40.94\left(2.997^{2}\right)\right)^{2}+((65.39)(2.997))^{2}}=2 \mathrm{~N}$
OR: $\quad r=\frac{\omega}{\omega_{n}}=\frac{2.997}{3.581}=0.837$
$\left|\left(f_{0}\right)_{\max }\right|=k X \sqrt{\left(1-r^{2}\right)+(2 \zeta r)^{2}}=525(0.00798) \sqrt{\left(1-(0.837)^{2}\right)+(2(0.223)(0.837))^{2}}=2 \mathrm{~N}$

The range of $f_{0}$ is as following: $-2 \mathrm{~N}<f_{0}<2 \mathrm{~N}$
2. Consider the system shown below, which consists of a uniform rigid bar $O A$ that is welded to a uniform disk. The composite system is pinned to ground at point 0 . Using the parameters indicated in the Figure 2, determine a) the equation of motion for the system in terms of the angular coordinate $\theta(t)$ and b ) the critical damping constant.


Figure 2
Sol.:

$$
\begin{aligned}
& F_{c 1}=c_{1} r \dot{\theta}, \quad F_{c 2}=3 c_{2} r \dot{\theta}, \quad F_{k}=k r \theta \\
& J_{0}=\frac{1}{2} m_{1} r^{2}+\frac{1}{3} m_{2}(3 r)^{2} \\
& \sum M_{0}=-J_{0} \ddot{\theta} \\
& k r^{2} \theta+c_{2}(3 r)^{2} \dot{\theta}+c_{1} r^{2} \dot{\theta}=-\left(\frac{1}{2} m_{1} r^{2}+3 m_{2} r^{2}\right) \ddot{\theta} \\
& \left(\frac{1}{2} m_{1}+3 m_{2}\right) r^{2} \ddot{\theta}+\left(c_{1}+9 c_{2}\right) r^{2} \dot{\theta}+k r^{2} \quad \theta=0 \\
& \left(\frac{1}{2} m_{1}+3 m_{2}\right) \ddot{\theta}+\left(c_{1}+9 c_{2}\right) \dot{\theta}+k \quad \theta=0 \\
& m_{e q}=\frac{1}{2} m_{1}+3 m_{2} \\
& c_{e q}=c_{1}+9 c_{2} \\
& k_{e q}=k \\
& c_{c r}=2 \sqrt{m_{e q} k_{e q}}=2 \sqrt{\left(\frac{m_{1}}{2}+3 m_{2}\right) k}
\end{aligned}
$$

3. Also late for a dinner party, Ahmed is driving his MAZDA on old Makkah road. Since Ahmed is a poor composer, he cannot afford to replace the bad shocks on his vehicle. They have lost most of their damping properties and behave essentially like springs.

Amanat Jeddah has halted old Makkah road maintenance for some reason. As a result, the road becomes very bumpy now. If the road surface can be approximately described by $y(t)=y_{0} \sin (\omega t)$,
(a) What is the most unfavorable speed for which Ahmed can travel?
(b) If Ahmed replaces the shocks on his MAZDA, is the most unfavorable speed less than, equal to, or greater than the most unfavorable speed found in part (a). Why?

Sol.:
a) Since the car have lost most of its damping properties:

$$
\zeta \lll 1 \Rightarrow \mathrm{c} \approx 0
$$



Road elevation: $y(t)=Y \sin (\omega t), \omega=\omega(v, l)$
Data: $m, k, l, v, Y$
$m \ddot{x}+k x=k y=k Y \sin (\omega t)$

Steady State Response: $x_{s \mathrm{ss}}=X \sin (\omega t-\Phi)$
$\left|\frac{X}{Y}\right|=\frac{1}{\left|1-r^{2}\right|}$
a) $v=\frac{l \omega}{2 \pi}, \quad$ Thus $v_{\text {unfavorable }}=\left.\frac{l \omega}{2 \pi}\right|_{\omega=\omega_{n}}=\frac{l \omega_{n}}{2 \pi}$
b) From Figure 3.15 page 240 (Rao) and knowing that $\omega_{n}$ is fixed for different value of $\zeta$.
Then, the speed after replacing the shocks is less than before, because if $\zeta$ increases the peak value is shifted to the left.

