

بسم الله الرحمن الرحيم

King Abdulaziz University
Engineering College
Department of Production and Mechanical System Design



MENG 470 Mechanical Vibrations

Final Exam
Closed-book Exam
Monday: 19/4/1425 H
Time Allowed: 120 mins



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Instructions

1. This is a closed book and closed notes Opportunity to Shine
2. Show all work for partial credit.

3. Assemble your work for each problem in logical order.
4. Justify your conclusion. I cannot read minds.

بسم الله الرحمن الرحيم

Mechanical Vibrations
MENG 470
Final Exam

Closed Book Exam
Time: 2 Hours
Monday: 19/4/1425 H

PART I

Select the most appropriate answer from the multiple choices given:

1. Which one of the following is a valid application for the Principle of Virtual Work?
 - a. Solve an eigenvalue problem for a system
 - b. Find the static equilibrium of a system
 - c. Find the dynamic equilibrium of a system
 - d. Determine the stability of a system
2. D'Alembert's Principle can be used directly to derive the differential equations of motion for a dynamic system.
 - a. True
 - b. False
3. A single-degree-of-freedom system has only one natural frequency.
 - a. True
 - b. False
4. The real part of the solution to the characteristic equation for a single-degree-of-freedom system is zero. Which one of the following best describes the system?
 - a. Underdamped
 - b. Unstable
 - c. Undamped
 - d. Nonperiodic
7. All of the normal modes of a multi-degree-of-freedom system are orthogonal, except for rigid body modes.
 - a. True
 - b. False
8. If a multi-degree-of-freedom system is positive semi-definite, which one of the following is not true?
 - a. The stiffness matrix is positive semi-definite
 - b. The eigenvalues of the system cannot be determined
 - c. The system has at least one rigid body mode
 - d. The system has at least one eigenvalue with a value of zero

9. If the modes, u , of a multi-degree-of-freedom system are normalized with respect to the mass matrix, $[m]$, the expression $[u]^T[m][u]$ yields the identity matrix.
- True
 - False
10. Which one of the following is not a characteristic of a continuous dynamic system?
- The equations of motion are partial differential equations
 - The system response cannot be calculated
 - The system has an infinite number of degrees of freedom
 - Both boundary conditions and initial conditions must be specified
11. Which one of the following is not required to perform a modal analysis of a continuous system?
- Calculate the natural frequencies
 - Calculate the natural modes
 - Normalize the natural modes
 - Calculate Rayleigh's Quotient
12. The flexibility and the stiffness matrices are the inverse of one another.
- True
 - False
13. The element stiffness matrices are always singular unless the boundary conditions are applied.
- True
 - False
14. The finite element method is:
- an approximate analytical method
 - a numerical method
 - an exact analytical method
15. What cause whirling of rotating shafts?
- Mass unbalance
 - Fluid friction in the bearings
 - Gyroscopic forces
 - All above.
16. To measure mechanical vibrations, we use:
- Accelerometers and signal analyzer.
 - Sound level meters.
 - Exciter and exciter controller.
 - All above.

17. The fundamental natural frequency of a system is a:
- The largest value
 - The smallest value
 - Any value.
18. When a two degree of freedom system is subjected to a harmonic force, the system vibrates at the:
- Frequency of applied force.
 - Smaller natural frequency
 - Larger natural frequency.
19. The response of an undamped system under resonance will be:
- Very large.
 - Infinity.
 - Zero.
20. Gibbs phenomenon denotes an anomalous behavior in the Fourier series representation of a:
- Harmonic function.
 - Periodic function.
 - Random function

PART II

- Q1. Consider the two-mass system shown in Figure 1. The physical parameters of the system are: $m_1=9$ kg, $m_2=1$ kg, $k_1=24$ N/mm, and $k_2=3$ N/mm.
- a) Determine the natural frequencies.
 - b) Derive modal equations for each natural frequency.
 - c) Determine the response of the system for the initial conditions $x_1(0)=1$ mm, $x_2(0)=0$ mm, and $\dot{x}_1(0)=\dot{x}_2(0)=0$ mm/s.
 - d) Decouple the equations of motion using *modal analysis*. Then determine the response of the system for the same initial conditions.
 - e) If a harmonic force, $F(t)=F_2 \cos(\omega t)$ is applied to the second mass. Derive frequency response functions from
 - (i) coupled systems.
 - (ii) uncoupled systems

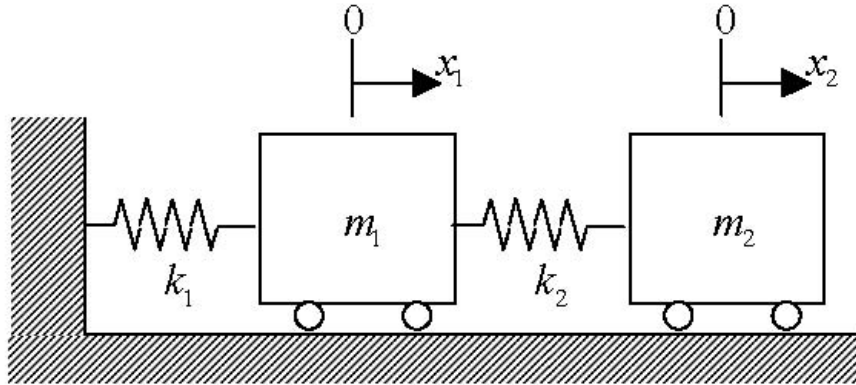


Figure 1

Q2. Engineer Abdulaziz designed one type of horizontal seismograph, a device that records earthquakes in the horizontal direction. It can be modeled as shown in Figure 2. The physical parameters of the seismograph are as following:

$$\begin{aligned} M &= 1 \text{ kg} & L &= 1 \text{ m} \\ m &= 4 \text{ kg} & a &= 0.2 \text{ m} \\ k &= 10 \text{ N/m} \end{aligned}$$

The acceleration of gravity is 9.81 m/s^2 , and the angle of oscillation is assumed to be small θ .

- Write the dynamic equations in matrix form.
- Calculate the natural frequencies and the modal matrix.
- As one of the first test cases to verify the appropriateness of the design, the engineer wants to investigate a strong impulse earthquake. When the seismograph was in equilibrium, assume that the ground moved to the right with an initial velocity of \dot{x}_0 . Decouple the equations and find resultant motions $x(t)$ and $\theta(t)$ each as a function of \dot{x}_0 and t .

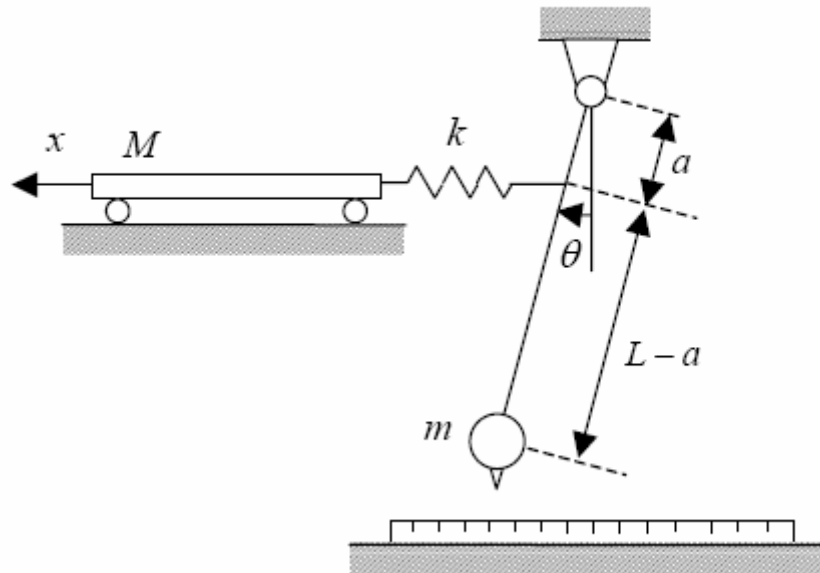


Figure 2

- Q3. A beam with constant mass and stiffness properties is fixed at $x=0$, and is supported by a linear spring at $x=L$ as shown in Figure 3.

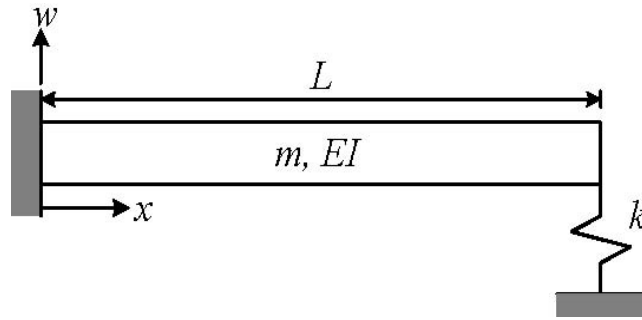


Figure 3

- Write the boundary conditions for the beam.
- Derive the characteristic equation for the system, but do not solve it.

Q4. A tapered rod is modeled as two uniform sections, as shown in Figure 4, where $EA_1=2EA_2$ and $m_1=2m_2$.

- a) Write the equations of motion in matrix form using FEM.
- b) Find the two natural frequencies of the longitudinal vibration
- c) Determine the corresponding mode shapes.
- d) Draw the mode shapes.

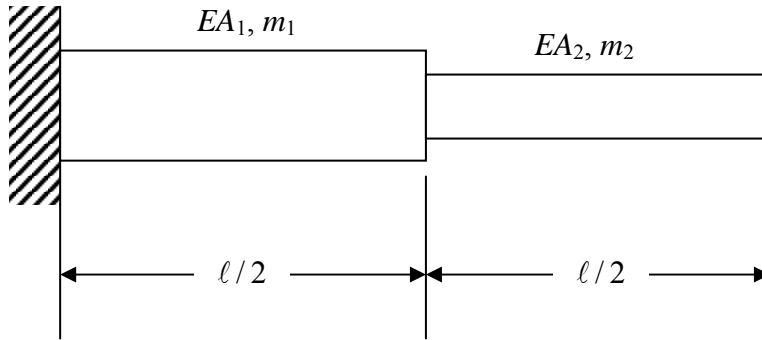


Figure 4

